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AN APPARATUS IN AN INFUSER FOR A LIQUID FOOD PRODUCT

TECHNICAL FIELD

The present invention relates to an apparatus in an infuser for a liquid food product, of the type which comprises a pressure vessel with an inlet located in its upper region for the product and an outlet located in its lower region for the product, the product inlet being disposed to finely divide the product entering into the pressure vessel into small droplets, and the infuser also including an inlet for steam, the inlet being disposed such that steam enters into the upper region of the pressure vessel through a concentric distributor chamber.

BACKGROUND ART

Heat treating food products for increased shelf-life is a well-known and often employed method in the art. The food products may, for example, be various dairy produce such as milk, cream or yoghurt. The heat treatment may take place in a plurality of ways, directly or indirectly. Indirect methods are, for example, heating using different types of heat exchangers. Of the direct methods there are two major groups, injection or infusion with steam. By employing a direct method, there will be obtained a very rapid heating, which is highly desirable today when, in order to improve the flavour properties, for example, in milk, the intention is many times to heat the product to elevated temperatures for a short period of time.

The present invention relates to an apparatus where use is made of infusion heating. Infusion entails heating a finely divided liquid in a steam chamber. The principle of heating a liquid, for example a liquid food, by injecting the liquid into a chamber filled with steam has been known from the early 19th century.

The first infusion heating plants were based on the same principle as condensers, with which excess steam from various chemical processes was condensed. The prior art plants thus included an infuser which consisted of a pressure vessel with an inlet for the product in the upper region of the pressure vessel. A distributor chamber discharged in the inlet, the distribution chamber finely dividing the incoming product into small, fine liquid droplets. In the bottom of the pressure vessel there is an outlet for the heated product. The pressure vessel was

further provided with an inlet for steam which, in these early infusers, was located in the lower region of the pressure vessel. The steam which, at a low level in the pressure vessel, is injected in towards the finely divided liquid droplets, causes the droplets to change direction. This was a disadvantage when these infusers were, for 5 example, employed for milk. Many of the droplets adhered to the hot inner walls in the infuser, where fouling by the product was the immediate consequence. Fouling entails major hygienic problems and is very difficult to wash off.

One way of obviating the problem inherent in fouling of product on the inner walls of the infuser is to place the steam inlet in the upper region of the infuser. The 10 steam is supplied from above over the distribution chamber of the product so that the droplets are not disturbed in their fall down through the steam chamber. But the admission of steam concurrently with the admission of product also has its drawbacks. This type of steam admission can give rise to fouling of the product around or in the distribution holes or distribution gaps which the distribution 15 chamber of the product inlet displays. This disturbs the heating of the product, the temperature of the product falls, in which event the steam pressure must be increased, and higher temperature differences will be obtained between product and steam. By designing the distribution holes so that they have a very thin edge out towards the steam chamber, this problem can be solved. By thus having extremely 20 thin fouling which is easily broken off, these foulings will disappear before they become disruptive.

But the concurrent admission of steam also involves problems with the incondensable gases which are always found in the product and which are given off in connection with its being heated. The steam also contains a part of these gases. 25 The gases may consist of oxygen, nitrogen, carbon dioxide and others. These gases must be continuously removed in order to be able to maintain the steam temperature at the same pressure. As a result of a concurrent steam admission, the incondensable gases mix more easily with the steam and it becomes more difficult to remove them without unnecessarily releasing steam from the process as well.

30 Swedish Patent Specification SE 513 414 describes an infuser which has the steam inlet placed low down in the infuser vessel. As a result, space is created for the incondensable gases to accumulate in the upper region of the vessel, from whence

they may be removed from the process in a simple manner. The steam inlet consists of a concentric aperture in the casing surface of the vessel. But this positioning of the steam inlet entails that hot steam enters at high speed into the vessel relatively close to the accumulation of product which is normally formed in the lower region of the vessel before the product departs from the vessel. Steam can then entrain with it product splashes on its way upwards in the vessel and undesirable fouling will be the result.

OBJECT OF THE INVENTION

One object of the present invention is to realise a steam inlet which obviates the majority of the negative effects displayed by prior art solutions and which is disposed such that the steam is guided so that it is at a suitable speed and direction when it meets the product droplets.

15 SOLUTION

This and other objects have been attained according to the present invention in that the apparatus of the type described by way of introduction has been given the characterising feature that the concentric distribution chamber is defined by the wall of the pressure vessel, a guide plate and at least one foraminated plate.

Preferred embodiments of the present invention have further been given the characterising features as set forth in the appended subclaims.

BRIEF DESCRIPTION OF THE ACCOMPANYING DRAWINGS

One preferred embodiment of the present invention will now be described in greater detail hereinbelow, with reference to the accompanying Drawings. In the accompanying Drawings:

Fig. 1 is a side elevation, partly in section, of an infuser according to the present invention;

Fig. 2 is a side elevation, partly in section, of the steam inlet of the infuser;

30 Fig. 3 is a plan view of a first foraminated plate; and

Fig. 4 is a plan view of a second foraminated plate.

The Drawings show only those parts and details essential to an understanding of the present invention.

DESCRIPTION OF PREFERRED EMBODIMENT

5 Fig. 1 shows an infuser according to the present invention. The infuser essentially consists of a conventional pressure vessel 1. The pressure vessel 1 has, in its upper region, an inlet 2 for the product which is to be treated in the infuser. The product inlet 2 terminates with a distribution chamber 3. The distribution chamber 3 is centrally positioned in the vessel 1 and is, on its lower wall 4, provided with a
10 large number of holes or gaps. The holes or gaps are intended to finely divide the incoming product so that this first forms fine jets which then progressively change into small, fine droplets which fall down through the pressure vessel 1.

In the lower region, preferably the lowermost region, of the pressure vessel 1 where the pressure vessel 1 forms a conical termination 5, there is disposed an outlet 15 6 for the product which has been heat treated in the infuser. The lower region of the conical termination 5 of the pressure vessel 1 normally functions as a holding tube, where the product stays at a certain predetermined temperature and during a certain predetermined period of time before being pumped further for additional treatment. The holding tube is indicated in Fig. 1 by the level line 7. Other types of holding 20 tubes can also be employed together with the above described infuser.

An infuser according to the present invention is further provided with at least, and in the preferred embodiment two, inlets 8 for steam. The steam inlets 8 are radially disposed and discharge in a concentric distribution chamber 9 which surrounds the entire pressure vessel 1. The concentric distribution chamber 9 with its 25 steam inlets 8 is placed on a lower level in the vessel 1 than the distribution chamber 3 of the product, but relatively high up in the pressure vessel 1. By distributing the incoming steam over two inlets 8, the speed of the steam is retarded and it is distributed more efficiently in the concentric distribution chamber 9. Both steam inlets 8 are preferably diametrically disposed.

30 The concentric distribution chamber 9 is defined, on the one hand, by the wall 10 of the pressure vessel 1 and, on the other hand, by a guide plate 11 and at least one foraminated, concentric plate 12. The guide plate 11 is, in the preferred

embodiment, double and in the interspace between the guide plates 12 there is the possibility of introducing air at room temperature in order to cool down the guide plates 11. Air is supplied through the inlet 13.

The foraminated, concentric plate 12 is placed below the inlets 8 for steam. In 5 the preferred embodiment, there are two foraminated plates 12 disposed above one another. Alternatively, there are three foraminated plates 12 placed above one another. The plates 12 may, as a result of the design of the pressure vessel 1, be of differing appearances, as is apparent from Fig. 3 and 4. Fig. 3 shows the upper, foraminated plate 12 and Fig. 4 the lower. Alternatively, and depending upon the 10 design of the pressure vessel 1, both plates 12 may have the appearance as shown in Fig. 3.

The foraminated plates 12 display a very large number of small holes 14 disposed therein. The holes 14 have a diameter of 1-10 mm, preferably 2-5 mm. The 15 apparatus according to the present invention has plates 12 which have 1000-1500 holes 14 each. The foraminated plates 12 should be sealingly secured against the casing wall 10 of the pressure vessel 1 and against the guide plate 11. If the anchorage is not completely tight, the incoming steam will take the path of least resistance into the pressure vessel 1, which normally implies that the steam will leak in at the side of the foraminated plates 12. The foraminated plates 12 should be 20 placed in spaced apart relationship from one another a distance which at least corresponds to the diameter of the holes 14 multiplied by five. This is in order to damp the turbulence of the steam after it has passed the upper, foraminated plate 12.

The guide plates 11 should have an extent below the foraminated plates 12 which at least corresponds to the diameter of the holes 14 multiplied by ten. 25 Preferably, the guide plates 11 have an extent below the foraminated plates 12 which is 50-60 mm in order to obtain as uniform a flow of steam as possible.

By the above described arrangement for admitting steam into the infuser, the speed and direction of the steam is controlled. When the steam finally enters into the pressure vessel 1, it should be at a speed of < 2 m/sec and should be directed 30 downwards so that it is sucked into towards and accompanies the product when this falls down through the pressure vessel 1.

The product coming into the infuser is pumped through a conduit to the inlet 2 in the upper region of the pressure vessel 1. The incoming product which, for example, may consist of milk, is normally at a temperature of 75-82°C, in certain applications up to 120°C. From the inlet 2 and the distribution chamber 3, the 5 product passes through a large number of holes or gaps provided in the lower wall 4 of the distribution chamber 3. The product thereby forms fine jets which progressively change into a large number of small droplets which fall freely through the pressure vessel 1 until they reach the liquid surface 7. After a short drop, downwardly directed steam will be sucked at a controlled speed in towards the 10 product droplets. The steam which is pressurised is at a temperature of approx. 120-170°C, preferably 140-150°C.

By condensation, the steam rapidly heats the product droplets to the desired temperature, and when the droplets reach the liquid surface 7, the product has reached the desired temperature. The distance between the distribution chamber 3 of the product and the liquid surface 7 should be so large that the product droplets have time to reach the desired temperature. The ready-heated product will thereafter, for a 15 certain period of time, stay in the conical termination 5 of the pressure vessel 1 before departing from the infuser through the outlet 6.

In that the admission of steam gives a controlled speed and direction for the 20 steam, the risk will be minimised that the steam disturbs the fall of the product droplets through the pressure vessel. In that the product droplets may fall undisturbed, the splashing of product is reduced and thereby the fouling of product on the walls 10 of the pressure vessel 1 is consequentially reduced.

During the heating process, incondensable gases are released, such as oxygen, 25 nitrogen and carbon dioxide, from the product. A certain quantity of incondensable gases will also come from the steam which is employed in the process. In that the coolest area in the infuser is disposed in the upper regions of the pressure vessel 1, the incondensable gases accumulate here and can be lead off from the infuser through the outlet 15. The apparatus according to the present invention, with steam 30 inlets 8 and the concentric distribution chamber 9 placed under the distribution chamber 3 of the product, and in that the steam enters into the pressure vessel 1 downwardly directed, creates space for the incondensable gases. The incondensable

gases can be accumulated in the upper region of the pressure vessel 1 with as little admixture of steam as possible. As a result, the steam consumption of the process is also minimised.

As will have been apparent from the foregoing description, the present invention realises an apparatus in an infuser which contributes in obtaining a steam admission into the infuser which is controlled in direction and speed in order to obtain an optimum product heating without the drawbacks which the prior art steam admission suffered from.